

Linear System Fundamentals

MEM 355 Performance Enhancement of Dynamical Systems

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System Representation



State Space & Transfer Function Representations

A linear time invariant system (LTI) can be represented by a system of first order differential equations:



Characteristic Equation

Recall, the characteristic equation of A is

$$\phi(\lambda) = |\lambda I - A| = \lambda^n + a_{n-1}\lambda^{n-1} + \dots + a_1\lambda + a_0$$

The roots of $\phi(\lambda)=0$ are the eigenvalues of A

The transfer function can be written

$$G(s) = C[sI - A]^{-1}B + D = C\frac{\operatorname{Adj}(A)}{|sI - A|}B + D = \frac{C\operatorname{Adj}(A)B + D|sI - A|}{|sI - A|}$$

So we see that in the SISO case

$$G(s) = \frac{n(s)}{d(s)}, \quad n(s) = \operatorname{C}\operatorname{Adj}(A)B + D|sI - A|, \quad d(s) = \phi(s)$$



The poles of G(s) are the eigenvalues of A!





Example: Steam Power Plant Control

Inputs		Outputs						
heat flow	Q	drump pressure	P_d				_9—	
water flow	$\omega_{_{e}}$	drum water level	ℓ		Drum	Valve		
throttle valve	A_t	steam flow	ω_{s}					
				Feedwater Valves				
Traditional desi	ign us	sed 3 SISO controlle	ers. T	The most				
difficult issue w	vas w	ith drum level contr	ol, pa	articularly				
at low load leve	els. Co	onsider 3 options:						
$\omega_{e} \rightarrow \ell$								
$\omega_e, A_t \to \ell, P_d$	ł				Downcome			
$\omega_e, A_t, Q \to \ell$	P_d, P_d	\mathcal{O}_{s}						
-								



Power Plant, 2

- N number of riser sections
- Ldo,L downcomer length and riser section length (total riser length/N)
- Ado,A downcomer, riser cross section areas
- wi mass flow rate at ith node
- Pi pressure at ith node
- Ti temperature at ith node
- si aggregate entropy at ith node
- vi specific volume at ith node
- wr,wdc,ws mass flow rates, riser, downcomer and turbine, respectively vdf,vdg drum specific volume, liquid and gas, respectively
- Pd drum pressure
- Td drum temperature
- V total drum volume
- Vw volume of water in drum
- xd net drum quality, xd=Vw/V
- ws0 throttle flow at rated conditions
- Pd0 drum pressure at rated conditions
- At normalized throttle valve position, at rated conditions At=1



Power Plant, 3

 $u_1 = q, u_2 = \omega_e, u_3 = A_t$ $\frac{\mathrm{d}\omega_{av}}{\mathrm{d}t} = f_1(\omega_{av}, s_1, s_2, s_3, P_{av}, P_d)$ $\frac{ds_1}{dt} = f_2(\omega_{av}, s_1, P_{av}) + g_{21}(P_{av}, s_1)u_1 + g_{22}(\omega_{av}, P_d)u_2$ $\frac{ds_2}{dt} = f_3(\omega_{av}, s_1, s_2, P_{av}) + g_{31}(P_{av}, s_2)u_1$ $\frac{ds_3}{dt} = f_4(\omega_{av}, s_2, s_3, P_{av}) + g_{41}(P_{av}, s_3)u_1$ $\frac{ds_4}{dt} = f_5(\omega_{av}, s_3, s_4, P_{av}) + g_{51}(P_{av}, s_4)u_1$ $\frac{dP_{av}}{dt} = f_6(\omega_{av}, s_1, s_2, s_3, s_4, P_{av}, P_d) + g_{61}(\omega_{av}, s_1, s_2, s_3, s_4, P_{av})u_1$ $\frac{dP_d}{dt} = f_7(\omega_{av}, s_1, s_2, s_3, s_4, P_{av}, P_d, V_w) + g_{71}(\omega_{av}, s_1, s_2, s_3, s_4, P_{av}, P_d, V_w) u_1 + g_{72}(P_d, V_w) u_2 - g_{73}(P_d, V_w) u_3$ $\frac{dV_W}{dt}$ $= f_8(\omega_{av}, s_1, s_2, s_3, s_4, P_{av}, P_d, V_w) + g_{81}(\omega_{av}, s_1, s_2, s_3, s_4, P_{av}, P_d, V_w) u_1 + g_{82}(P_d, V_w) u_2 - g_{83}(P_d, V_w) u_3 + g_{83}(P_d, V_w) u_3 +$ $y_1 = P_d$, $y_2 = = h_2(V_w)$, $y_3 = \omega_s = h_3(P_d) + d_3(P_d)u_3$



Linearized Dynamics, Poles

Poles as a function of load level, 5%-100%





Linearized Dynamics, Zeros

 $\omega_e \to \ell$





Transmission Zeros

 $\omega_e, A_t \to P_d, \ell$





Transmission Zeros

 $\omega_e, A_t \to P_d, \ell$





Transmission zeros

 $Q, \omega_e, A_t \to P_d, \ell, \omega_s$





Transmission zeros

 $Q, \omega_e, A_t \to P_d, \ell, \omega_s$





real

Resolvant Matrix and State Transition Matrix

The matrix $[sI - A]^{-1}$ is called the "resolvent" matrix.

It's inverse Laplace Transform is the "state transition matrix":

$$\Phi(t) = \mathcal{L}^{-1}\left(\left[sI - A\right]^{-1}\right) = e^{At}$$
In this later

In terms of Φ we can write

State response

$$\mathbf{x}(t) = \Phi(t) x_0 + \int_0^t \Phi(t-\tau) B u(\tau) d\tau$$

$$y(t) = C\Phi(t)x_0 + C\int_0^t \Phi(t-\tau)Bu(\tau)d\tau + Du(t)$$

Output response



A Stability Lemma

Lemma: <u>Assume the system poles, i.e., the eigenvalues of the *A* matrix are all in the strict left complex plane, then we have the following: (1) The response due to the initial state is,</u>

and $\lim_{t \to \infty} \Phi(t) x_0 = 0$ (2) The response to the input, u(t) ($x_0 = 0$) $y(t) = C \int_0^t \Phi(t - \tau) B u(\tau) d\tau + D u(t)$ is bounded for every bounded input u(t).



Stability Definitions

Definition: A linear time-invariant system is <u>BIBO</u> (Bounded-Input Bounded-Output) stable if and only if every bounded input results in a bounded output.

Definition: A linear time-invariant system is <u>internally</u> stable if the solution x(t) of $\dot{x}(t) = Ax(t), \quad x(0) = x_0$

• tends toward zero as $t \to \infty$ for arbitrary x_0 .

Aren't these the same? ----- No!



Stability Theorems

Theorem: An LTI system with transfer function G(s) is BIBO stable if and only if the poles of G(s) are strictly in the left half plane.

Theorem: An LTI system with state space parameters A, B, C, D is internally stable if and only if the all of the eigenvalues of A are strictly in the left half plane.

Note: Internal stability is a stronger condition than BIBO stability. BIBO stability only reflects the attributes of the system that are observable from the output and controllable from the input. There may be hidden modes that are unstable. Much more about this

later



Similarity Transformations

$$\begin{split} \dot{x} &= Ax + Bu \\ y &= Cx + Du \end{split} \quad x \in R^n, u \in R^m, y \in R^p \qquad \dot{x} = Ax + bu \\ y &= cx + du \end{split} \\ \text{Now consider the transformation to new states } z, defined by \\ x &= Tz \Leftrightarrow z = T^{-1}x \\ T\dot{z} &= ATz + Bu \\ y &= CTz + Du \end{cases} \Rightarrow \begin{aligned} \dot{z} &= T^{-1}ATz + T^{-1}Bu \\ y &= CTz + Du \end{aligned} \Rightarrow \begin{aligned} y &= CTz + Du \\ y &= CTz + Du \end{aligned}$$
 so that, $\dot{z} &= A^*z + B^*u \\ y &= C^*z + D^*u \end{aligned} \quad A^* &= T^{-1}AT, \ B^* &= T^{-1}B, \ C^* &= CT, \ D^* = D \end{aligned}$



Diagonal Form

eigen-system
of A:
$$\lambda_1 \quad \lambda_2 \quad \cdots \quad \lambda_n \quad \leftarrow$$
 eigenvalues
 $h_1 \quad h_2 \quad \cdots \quad h_n \leftarrow$ independent eigenvectors
 $T \triangleq [h_1 \quad h_2 \quad \cdots \quad h_n]$
 $\Rightarrow A^* = [h_1 \quad h_2 \quad \cdots \quad h_n]^{-1}A[h_1 \quad h_2 \quad \cdots \quad h_n]$
 $= \begin{bmatrix} \lambda_1 \quad 0 \quad \cdots \quad 0 \\ 0 \quad \lambda_2 \quad \ddots \quad \vdots \\ \vdots \quad \ddots \quad \ddots \quad 0 \\ 0 \quad \cdots \quad 0 \quad \lambda_n \end{bmatrix}$

$$\dot{z}_i = \lambda_i z_i + b_i^* u$$
, $i = 1, \dots, n$

A decoupled system of
$$n 1^{st}$$
 order ode's



Companion Form

Consider the single-input system:

 $\dot{x} = Ax + bu$

and the transformation

$$\begin{bmatrix} b & Ab & \cdots & A^{n-1}b \end{bmatrix}^{-1} = \begin{bmatrix} q_1 \\ q_2 \\ \vdots \\ q_n \end{bmatrix}, \quad T^{-1} = \begin{bmatrix} q_n \\ q_nA \\ \vdots \\ q_nA^{n-1} \end{bmatrix}$$

Apply the similarity transform to obtain the system:

$$\dot{z} = \begin{bmatrix} 0 & 1 & 0 \\ & \ddots & \ddots & \\ & & 0 & 1 \\ -a_0 & -a_1 & \cdots & -a_{n-1} \end{bmatrix} z + \begin{bmatrix} 0 \\ \vdots \\ 0 \\ 1 \end{bmatrix} u$$



Proof (1)

We proceed in two steps: First establish b^* , then A^*

$$\mathcal{C}^{-1}\mathcal{C} = I \Rightarrow \begin{bmatrix} q_1 \\ q_2 \\ \vdots \\ q_n \end{bmatrix} \begin{bmatrix} b & Ab & \cdots & A^{n-1}b \end{bmatrix} = I$$
$$\Rightarrow q_n \begin{bmatrix} b & Ab & \cdots & A^{n-1}b \end{bmatrix} = \begin{bmatrix} 0 & \cdots & 0 & 1 \end{bmatrix}$$
$$\Rightarrow b^* = T^{-1}b = \begin{bmatrix} 0 \\ \vdots \\ 0 \\ 1 \end{bmatrix}$$



Proof (2)

$$T^{-1}T = I \Rightarrow \begin{bmatrix} q_n \\ q_n A \\ \vdots \\ q_n A^{n-1} \end{bmatrix} T = I \Rightarrow \begin{bmatrix} q_n T \\ q_n A T \\ \vdots \\ q_n A^{n-1} T \end{bmatrix} = \begin{bmatrix} 1 & 0 & \cdots & 0 \\ 0 & 1 & \ddots & \vdots \\ \vdots & \ddots & \ddots & 0 \\ 0 & \cdots & 0 & 1 \end{bmatrix}$$

$$T^{-1}AT = \begin{bmatrix} q_n \\ q_n A \\ \vdots \\ q_n A^{n-1} \end{bmatrix} AT = \begin{bmatrix} q_n AT \\ q_n A^2 T \\ \vdots \\ q_n A^n T \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ \ddots & \ddots & \\ 0 & 0 & 1 \\ Y_1 & Y_2 & \cdots & Y_n \end{bmatrix}, Y = q_n A^n T$$

To compute *Y*, suppose det $(\lambda I - A) = \lambda^n + a_{n-1}\lambda^{n-1} + \dots + a_0$. note C-H Theorem $\Rightarrow A^n + a_{n-1}A^{n-1} + \dots + a_0I = 0$ $Y = q_n A^n T = q_n (-a_{n-1}A^{n-1} - \dots - a_0I)T = -a_{n-1}q_n A^{n-1}T - \dots - a_0q_nT = [-a_0 - a_1 - \dots - a_{n-1}]$



SISO Companion Forms





State Feedback Pole Placement

Given a linear system:

 $\dot{x} = Ax + Bu$

find a state feedback control:

u = Kx

such that the closed loop system:

 $\dot{x} = Ax + BKx = (A + BK)x$

has a specified (self-conjugate) set of poles $\{p_1, p_2, ..., p_n\}$.



Pole Placement Sol'n: SISO Case

• Convert $\dot{x} = Ax + bu$ to controller form (phase variable form) using x = Tz:

$$\dot{z} = \begin{bmatrix} 0 & 1 & 0 \\ \ddots & \ddots & \\ & 0 & 1 \\ -a_0 & -a_1 & \cdots & -a_{n-1} \end{bmatrix} z + \begin{bmatrix} 0 \\ \vdots \\ 0 \\ 1 \end{bmatrix} u$$
controller form
$$\mathbf{Set} \ u = \begin{bmatrix} k_1 & k_2 & \cdots & k_n \end{bmatrix} z \text{ and obtain closed loop: } \dot{z} = \begin{bmatrix} 0 & 1 & 0 \\ \ddots & \ddots & \\ & 0 & 1 \\ k_1 - a_0 & k_2 - a_1 & \cdots & k_n - a_{n-1} \end{bmatrix} z$$

• Expand desired closed loop characteristic polynomial and compare coefficients, and solve for k_1, \ldots, k_n :

 $\phi_{cl}(\lambda) = (\lambda - p_1)(\lambda - p_2)\cdots(\lambda - p_n) = \lambda^n + \alpha_{n-1}\lambda^{n-1} + \cdots + \alpha_0 \Rightarrow \alpha_0 = a_0 - k_1, \alpha_1 = a_1 - k_2, \dots, \alpha_{n-1} = a_{n-1} - k_n$





Pole Place Design: The Easy Way

PLACE Pole placement technique

K = PLACE(A,B,P) computes a state-feedback matrix K such that

the eigenvalues of A-B*K are those specified in vector P. No eigenvalue should have a multiplicity greater than the number of inputs.





Example: XV-15 Hover Dynamics



u, body *x* – velocity *w*, body *z* – velocity *q*, pitch rate θ pitch angle

 δ_{loc} rotor longitudinal cyclic pitch δ_{co} collective pitch





Example, XV15 Longitudinal Modes

λ	-0.4280	-0.1915	$0.1314 \pm j0.3084$
U	0.9996	0.2017	$0.9999 \pm j0.0000$
W	-0.0243	-0.9795	$-0.0110 \pm j0.0052$
q	-0.0054	-0.0002	$0.0024 \pm j 0.0026$
θ	0.0127	0.0012	$-0.0044 \pm j0.0097$



Example, XV-15 Longitudinal Stabilizer



Old eigenvalues:

$$p_{old} = \{-0.4280, -0.1915, -0.1314 \pm j0.3084\}$$

Choose new eigenvalues:

$$p = \{-0.5, -0.2, -0.25 \pm j0.25\}$$



XV-15 Pole Placement





Routh-Hurwitz Stability Criterian

- Given a polynomial that represents the pole polynomial of a transfer function or the characteristic polynomial of square matrix it is very easy to determine the roots (and therefore assess stability) using numerical computations – providing the coefficients are all specified!
- But suppose one or more of the coefficients are not specified. Rather, we want to determine an admissible range of values such that the system is stable. That is the control designer's problem and where the Routh-Hurwitz criterion is useful.



Routh-Hurwitz –2

Theorem: Consider the polynomial

$$p(s) = s^{n} + a_{n-1}s^{n-1} + \dots + a_{1}s + a_{0}$$

A <u>necessary</u> condition that all roots are strictly in the left half plane is that all coefficients are strictly positive.

Note that this provides only a necessary condition. To determine a necessary and sufficient condition we assemble the Routh array.



Routh-Hurwitz–3

Theorem: Consider the polynomial

$$p(s) = s^{n} + a_{n-1}s^{n-1} + \dots + a_{1}s + a_{0}$$

The associated system is (BIBO or internally) stable if and only if all elements of the first column of the Routh array are strictly positive.

If the polynomial is the 'minimal' transfer function denominator If the polynomial is the Characteristic polynomial of the A matrix



Routh Array

 $p(s) = s^{n} + a_{n-1}s^{n-1} + \dots + a_{1}s + a_{0}$

Construct a matrix as shown by with the first two rows using the using the above polynomial coefficients.

Then construct the remaining rows





Routh-Hurwitz – Example 1

 $p(s) = s^{5} + 15s^{4} + 74.25s^{3} + 121s^{2} + 20Ks + 2K$

s^{5}	1	74.25	20 <i>K</i>
<i>s</i> ⁴	15	121	2 <i>K</i>
s^3	65.9	19.86 <i>K</i>	
s^2	121 - 4.52K	2 <i>K</i>	
s ¹	$\frac{2271K - 89.76K^2}{121 - 4.52K}$		
s^{0}	2 <i>K</i>		

 $121 - 4.52K > 0, 2271K - 89.76K^2 > 0, K > 0$



$$K < \frac{121}{4.52} = 26.769 \qquad K < \frac{2271}{89.76} = 25.308$$

Routh-Hurwitz – Example 2 $p(s) = s^4 + (1+K)s^3 + (1+6K)s^2 + 10Ks + 8K$



$$1+K > 0$$
, $1-3K+6K^2 > 0$, $1-23K+26K^2 > 0$, $K > 0$